

## TWIND Summer School

# Floating Offshore Wind Design solutions and Higher-order hydrodynamics

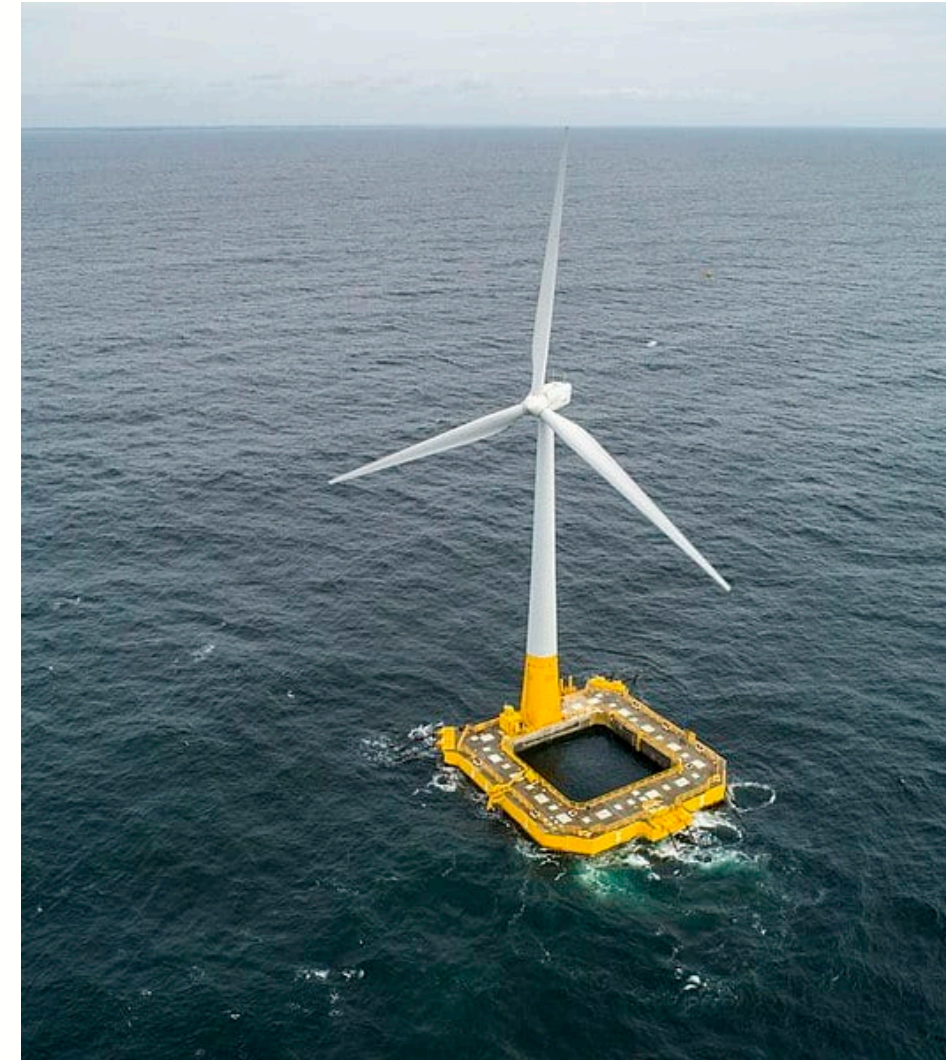
6/07/2021

Daniel Milano

The offshore wind industry transitioning from fixed to floating, following oil & gas footsteps

Why go floating?

- **Access to deep-water sites:** access to large areas with water depths of 60 m or more
  - Strong and steady winds
  - Proximity to densely populated coastal areas
- **Design:** alternative to bottom-fixed in mid-depth sites (30 - 60 m)
- **Construction and operation:** possibility of assembly on-quay and easy transport using towing vessels
- **Bathymetry sensitivity:** less sensitive to local constraints, mooring system is the only element in direct contact with seabed
- **Social and visual impact**



Ideol Floatgen Floating Wind Turbine



- Design
- Challenges



Fukushima FORWARD | 5MW, Osaka 2016  
Photo: Yumiuri Shimbun

- Floater and transition piece: buoyancy and structural support
- Mooring and anchoring system: station keeping
- Dynamic power cable: grid connection

Trade-off between stability and costs depending on floater's size

- Small draft and small diameter: lower hydrostatic stability
- Large draft and diameter: higher manufacturing costs

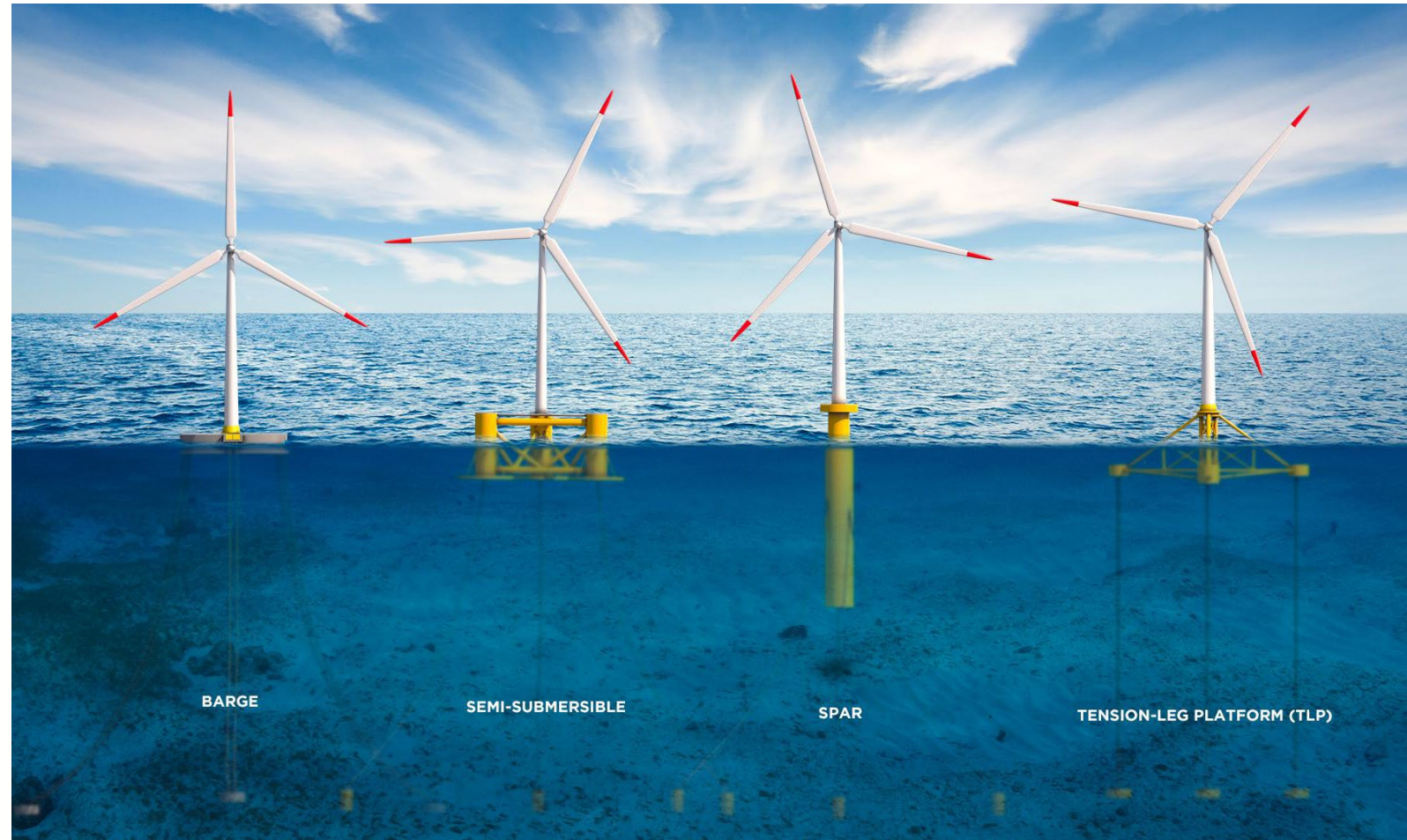


Photo: Green Giraffe



Geometry of the floating platform impacts motion response and structural behaviour of FOWTs

- **Slender spar buoys**  
(large draft, small diameter)
  - Prone to fatigue
- **Barge structures**  
(small draft, large diameter)
  - Large heave and pitch motion responses
- **Tension-leg platform**  
(small draft, small diameter)
  - Reduced motion responses due to stiff, pre-tensioned mooring system

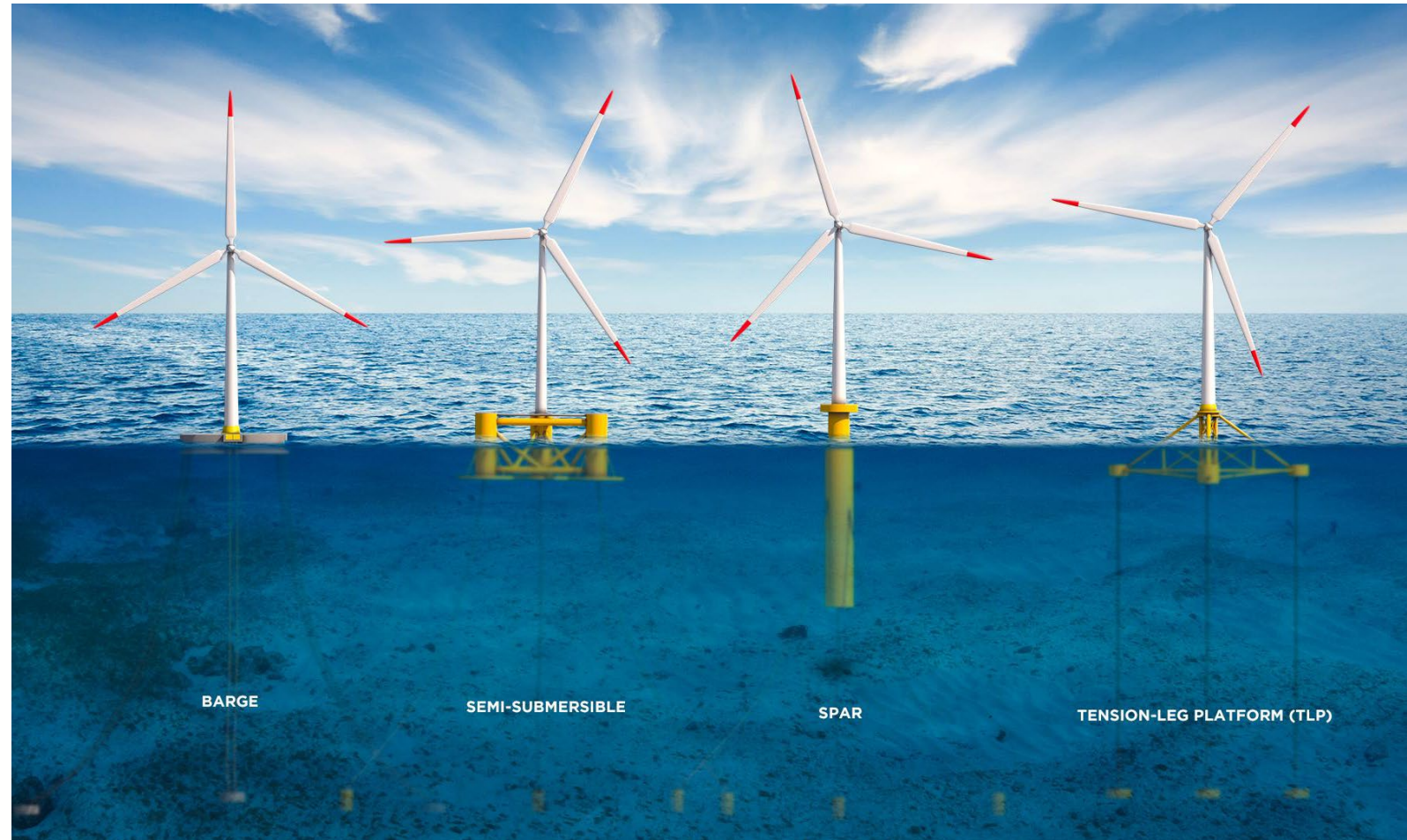


Photo: Green Giraffe

- Servo – aerodynamics (Blade element - momentum theory)
  - Blade loading
  - Control system (blade pitch, yaw, torque)
  - Drivetrain & Power generation
- Hydrodynamics (Potential flow theory + Morison quadratic drag)
  - Wave excitation: Froude-Krylov
  - Incident-wave scattering: diffraction
  - Wave acceleration forces: added mass
  - Radiation damping
  - Wave velocity forces: drag
  - Hydrostatic restoring forces
- Mooring system (quasi-static or dynamic)
  - Station keeping forces



Umaine 15 MW VoltturnUS Semi-sub

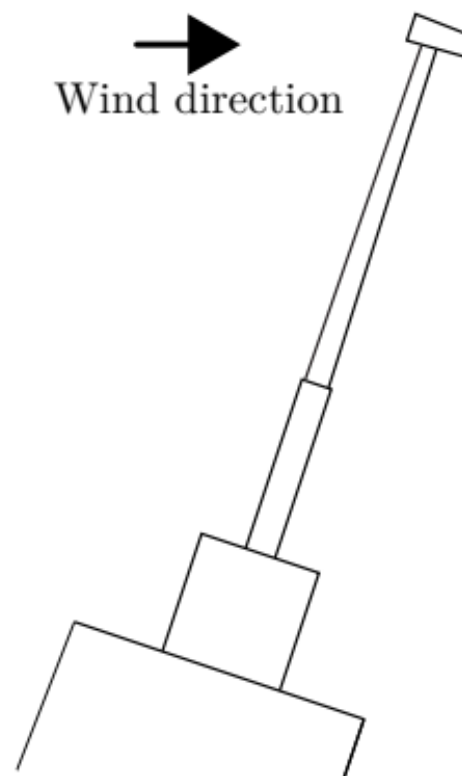
## Wind-induced motion response for TLP wind turbine designs

### Pitching

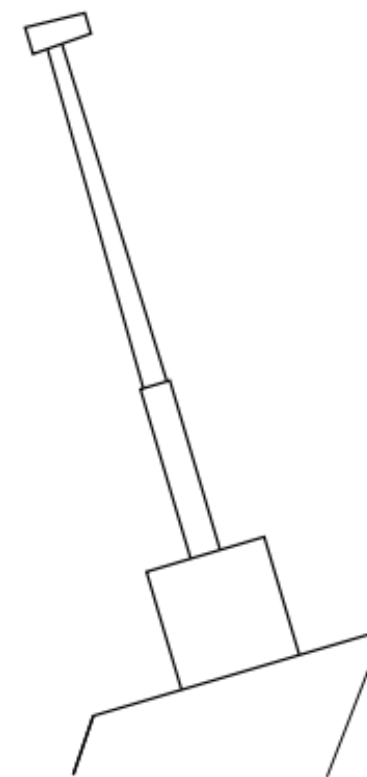
- Positive keel-to-nacelle vector component following wind
- Rotor leans away from the wind

### Counter-pitching:

- Negative keel-to-nacelle vector component following wind
- Rotor leans towards the wind



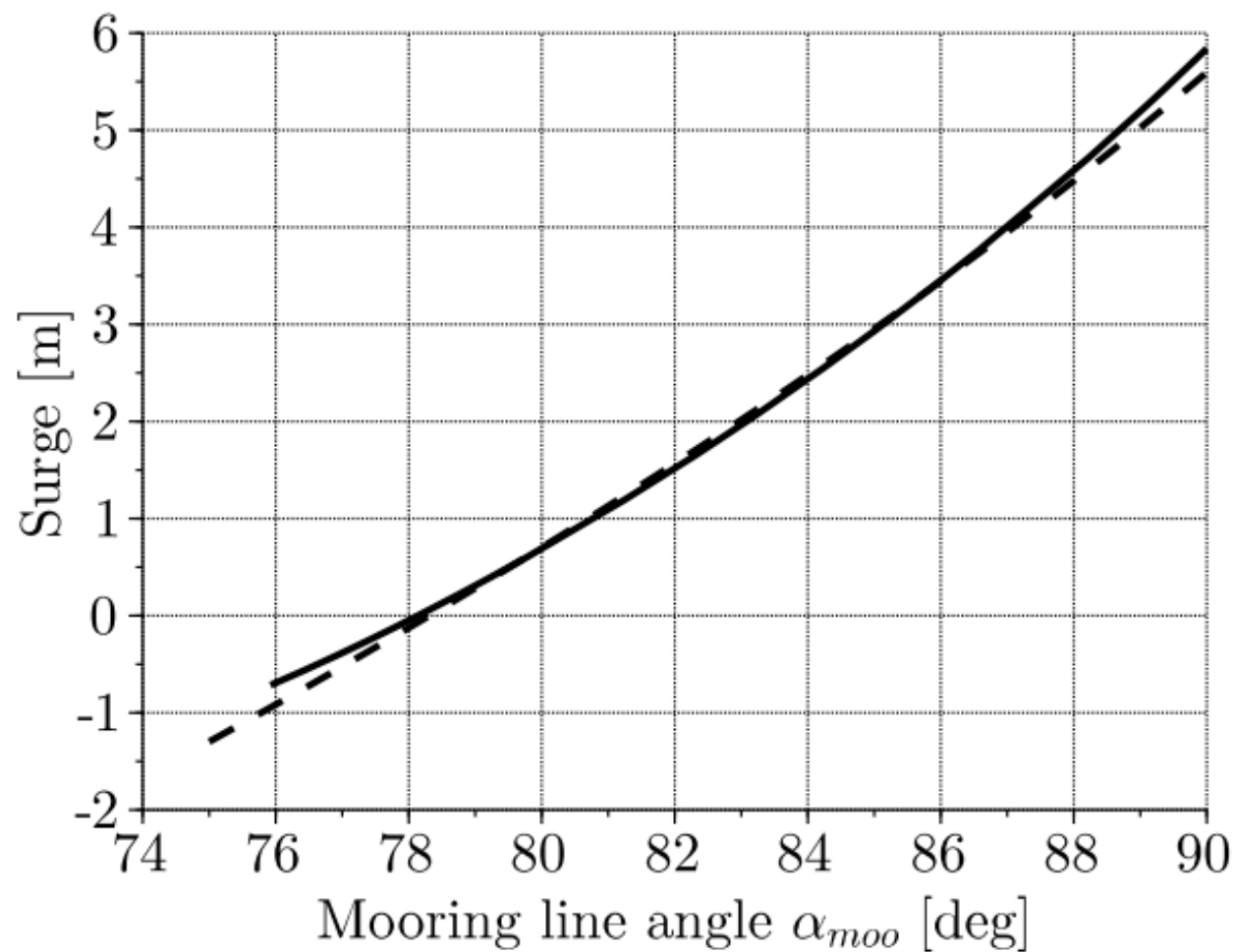
(a) Pitching



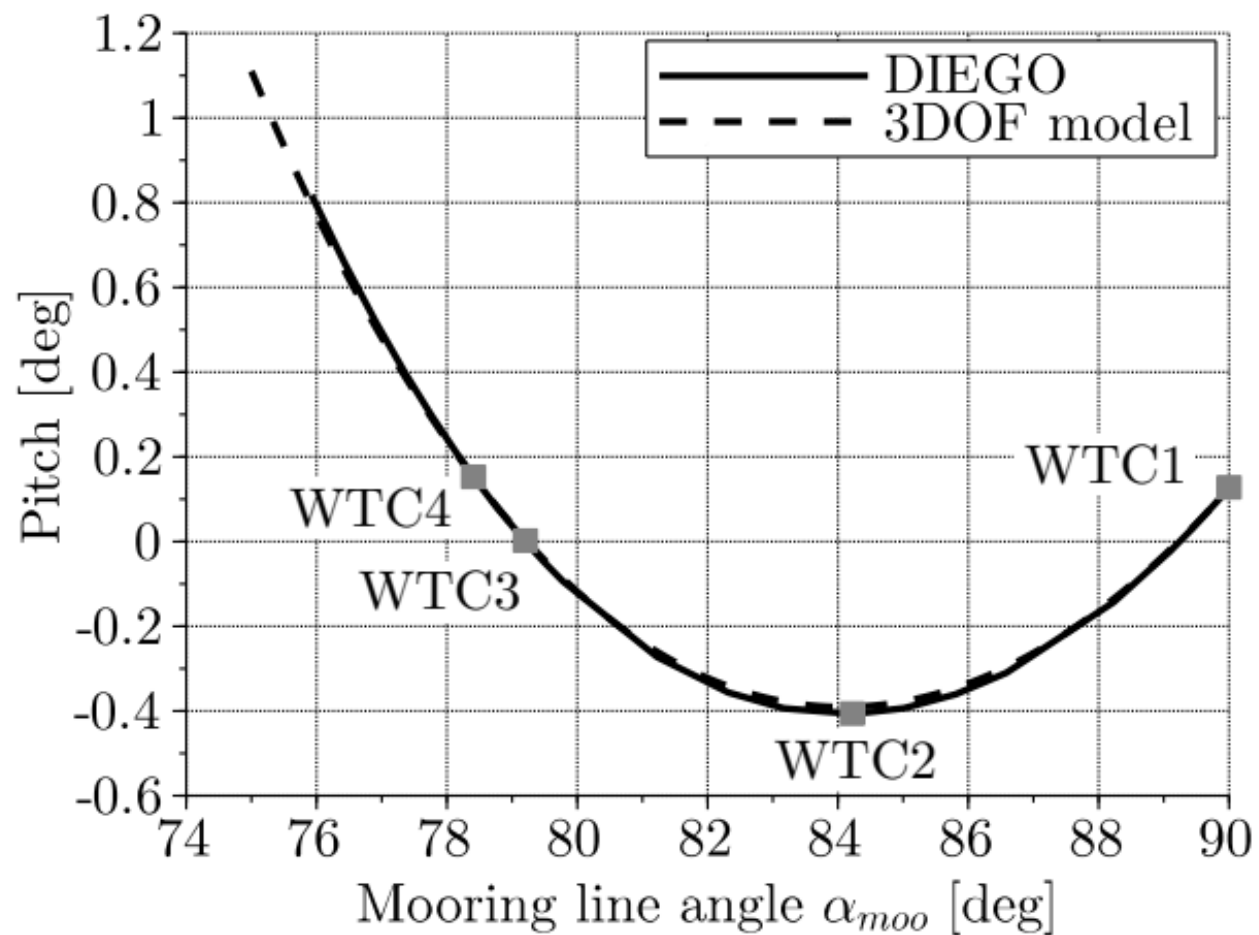
(b) Counter-pitching

Pitching and counter-pitching wind response

## Example: pitching & counter-pitching FOWTs



(a) Surge displacement

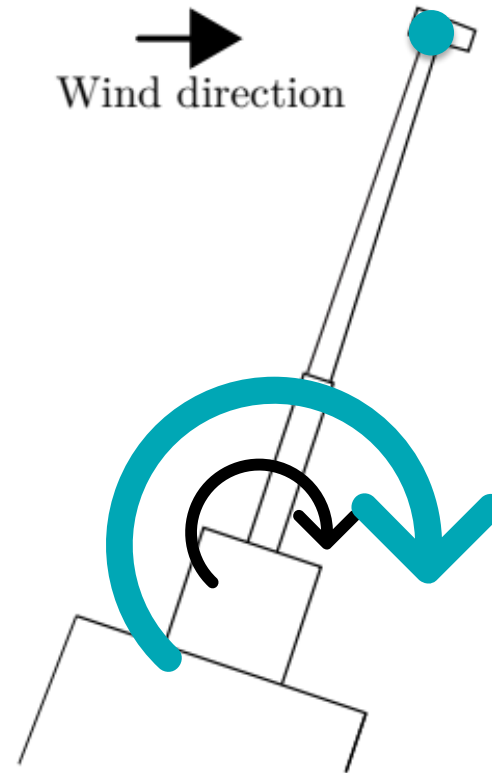


(b) Pitch displacement

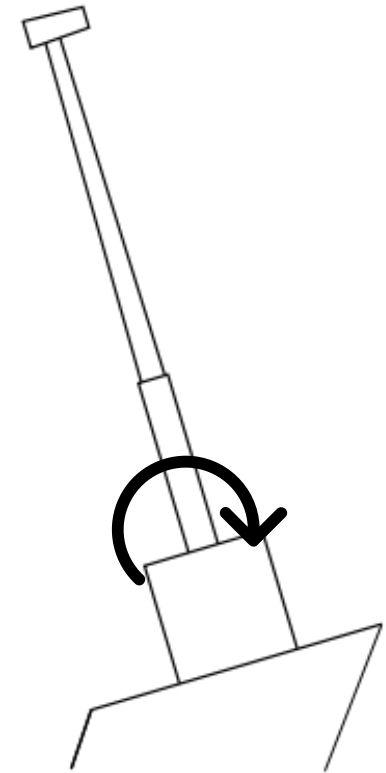
Pitching response at varying tendon inclination



- A keeling wind turbine is subject to gravity forces
- Bending moments at tower bottom



(a) Pitching

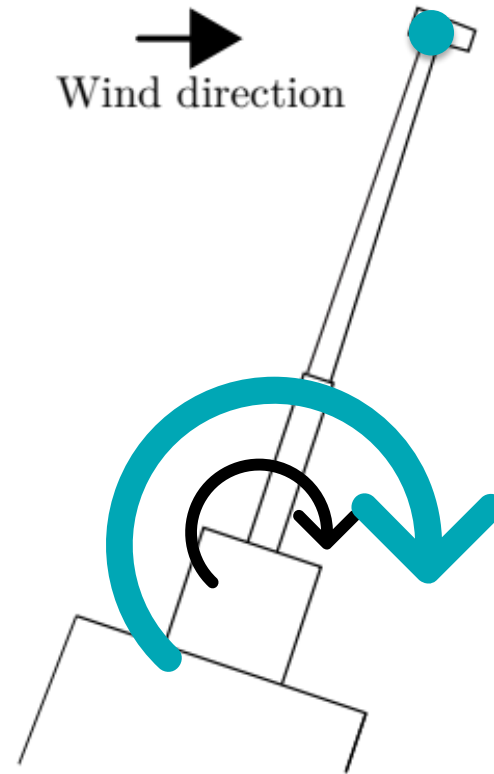


(b) Counter-pitching

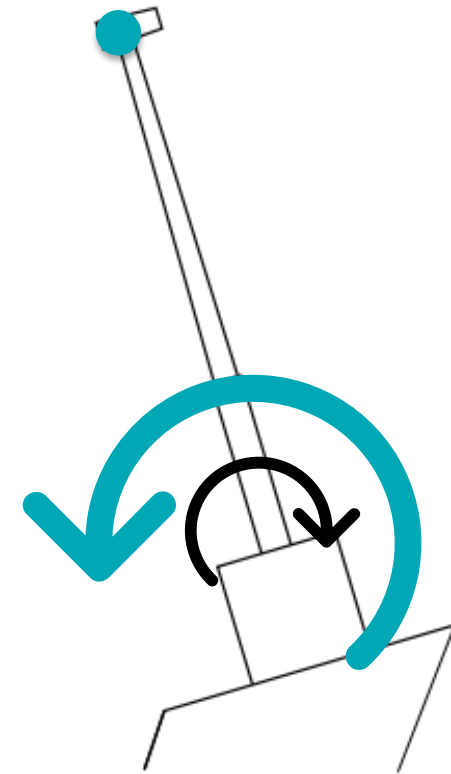
Pitching and counter-pitching wind response

- A keeling wind turbine is subject to gravity forces
- Bending moments at tower bottom
- In counter-pitching configuration, wind heeling moment can help offset gravity-based tilting moment

TLPs are better, right?



(a) Pitching



(b) Counter-pitching

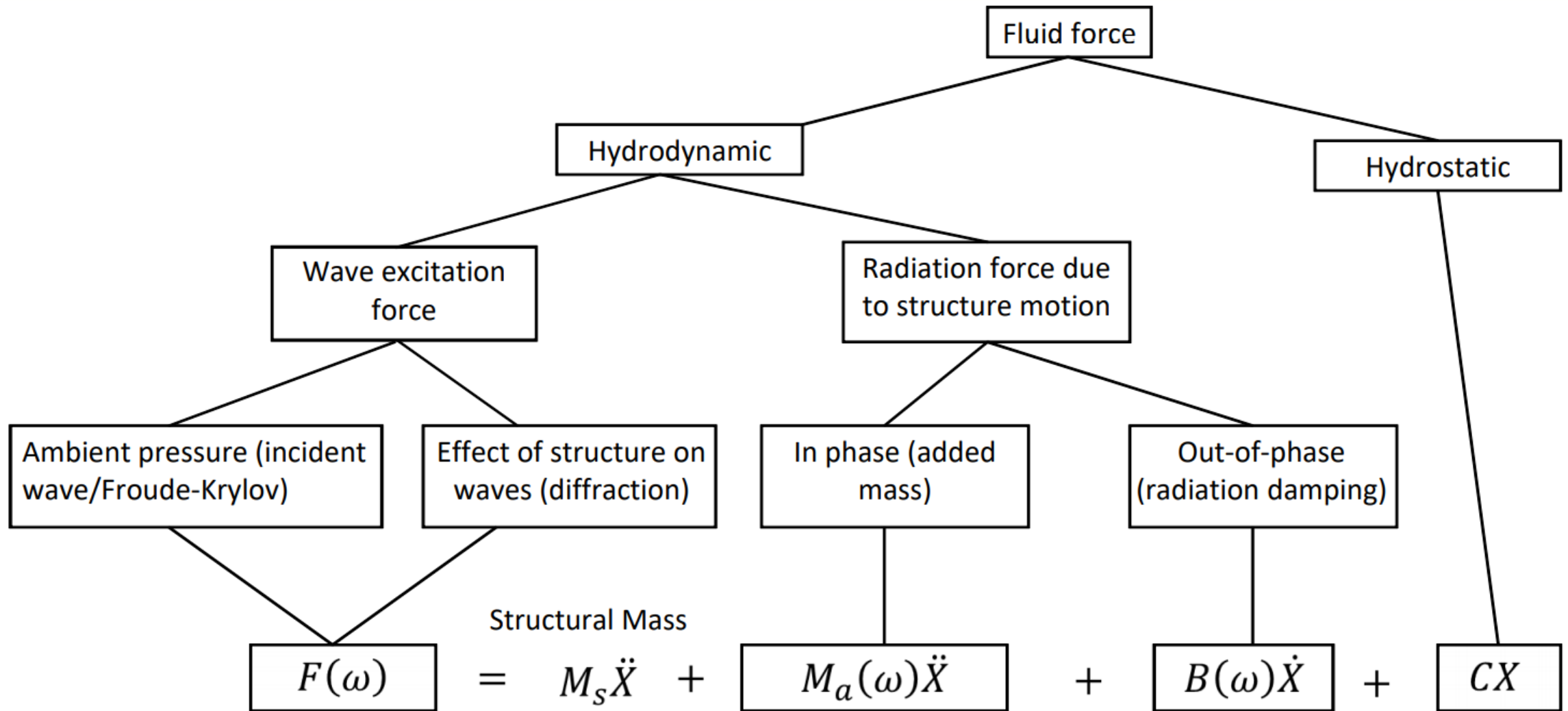
Pitching and counter-pitching wind response

- Design
- Challenges



Ocean Valiant drilling platform | Shetland, 2016  
Photo: gCaptain





Equation of motion

Wave elevation

$$\eta^{(1)}(t) = \sum_i A_i \cos(k_{x,i} x + k_{y,i} y - \omega_i t + \varphi_i)$$

Linear wave load

$$F_{ID}^{(1)}(t) = \sum_i A_i F e_i \cos(k_{x,i} x + k_{y,i} y - \omega_i t - \theta_i + \varphi_i)$$

- Motion-agnostic

- TLPs plane motions (heave, roll and pitch) in the order of 2 - 5 sec
- Low amount of wave energy carried in this frequency interval
- Limited exciting forces obtained from linear diffraction-radiation models
- **However**, experimental studies have shown:
  - High-frequency resonant responses at double the wave periods, in the 4 - 10 sec interval
  - Slow drift forces, similar to an additional sea current



Froude-Krylov and Diffraction forces  
1<sup>st</sup> order

## Wave elevation

$$\eta^{(1)}(t) = \sum_i A_i \cos(k_{x,i} x + k_{y,i} y - \omega_i t + \varphi_i)$$

## Linear wave load

- Motion-agnostic

$$F_{ID}^{(1)}(t) = \sum_i A_i F e_i \cos(k_{x,i} x + k_{y,i} y - \omega_i t - \theta_i + \varphi_i)$$

## Quadratic wave load

- Motion-dependent
- Enhanced with FOWTs

$$F^{(2)}(t) = \sum_i \sum_j A_i A_j QTF^+(\omega_i; \omega_j)$$



$$\cdot \cos[(k_i + k_j) x - (\omega_i + \omega_j) t + (\varphi_i + \varphi_j) + \varphi(\omega_i; \omega_j)_{QTF+}]$$

$$+ \sum_i \sum_j A_i A_j QTF^-(\omega_i; \omega_j)$$



$$\cdot \cos[(k_i - k_j) x - (\omega_i - \omega_j) t + (\varphi_i - \varphi_j) + \varphi(\omega_i; \omega_j)_{QTF-}]$$

Froude-Krylov and Diffraction forces  
1<sup>st</sup> and 2<sup>nd</sup> order





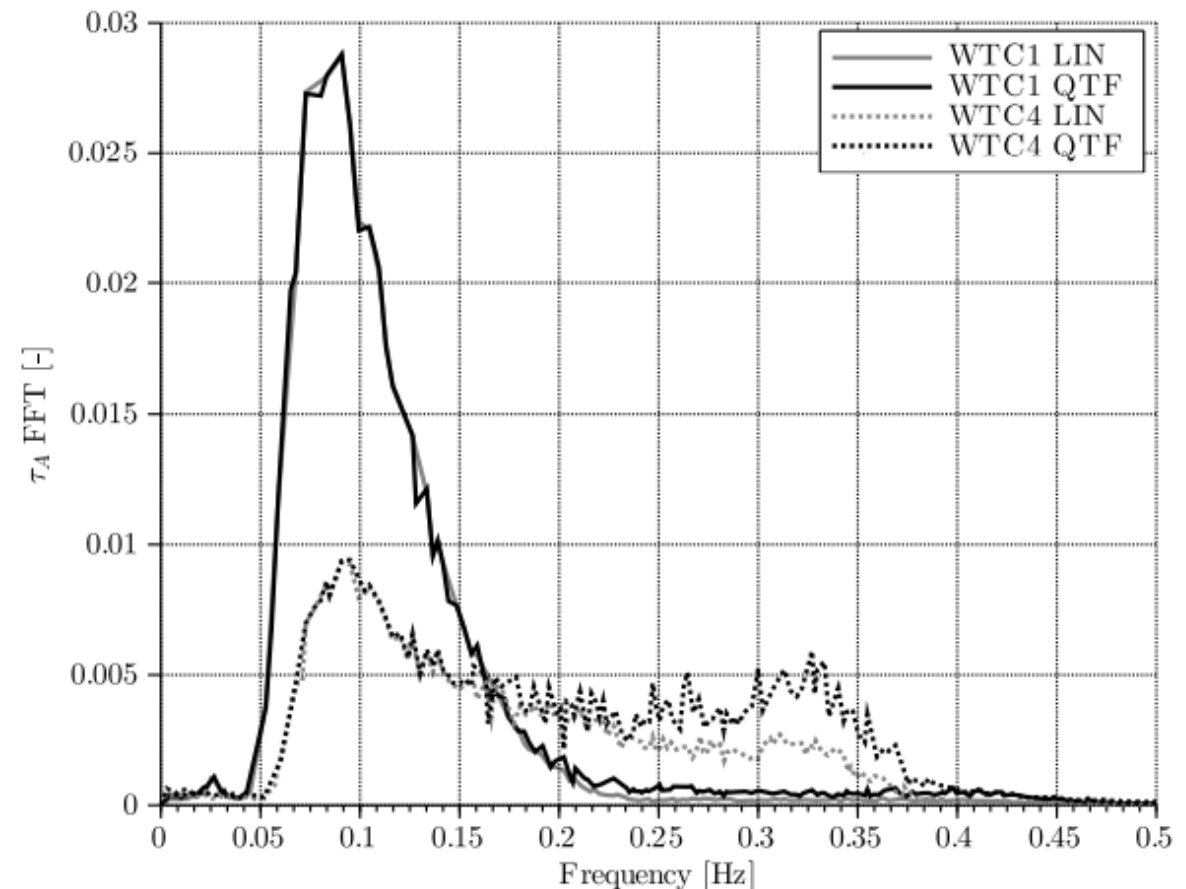
## Springing (TLP only)

- High-frequency resonant effect
- Observed when the peak period of the sea state is approximately twice the structure's natural period in heave, pitch or roll
- Mainly impacts TLP structures on tendon fatigue consumption, a key design aspect



## Slow-drift forces (all FOW designs)

- Low-frequency loads
- Observed in the form of an averaged horizontal force similar to a sea current



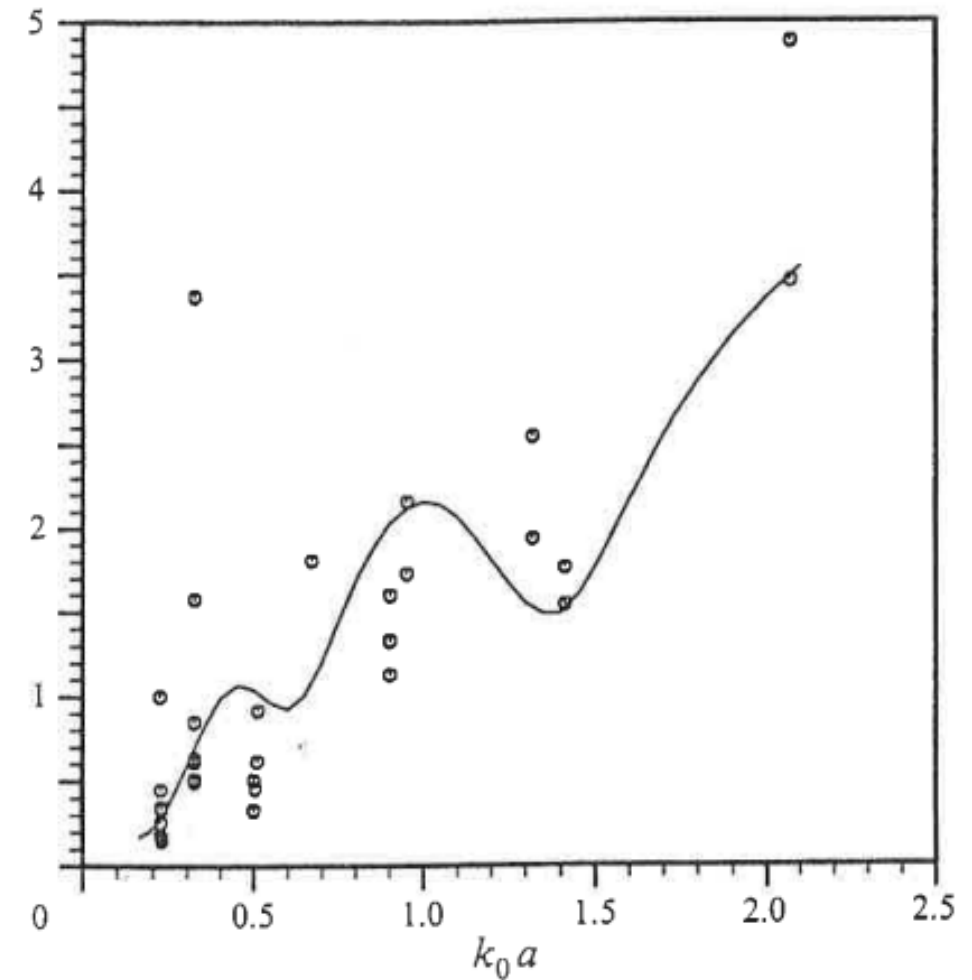
Tension frequency response to irregular sea state, DTU-TLPWT config. WTC1 and WTC4.  $H_s = 10.74\text{m}$ ,  $T_p = 12.4\text{s}$

- Deep-water TLPs and gravity-base foundations (GBF) have shown resonant effects in natural period range of 3 - 5 sec
- Observed in basin tests and at sea when sea state peak periods were 3 - 5 times the resonant periods of affected structure
  - Sea state peak period 9 – 15 sec



## Ringing (TLP only)

- Highly non-linear, third-order resonant effect
- Strong forces in very short period of time, causing impulse-like excitations
- Sporadic occurrence during extreme sea states, associated with high and steep waves



Modulus of third-harmonic horizontal force divided by  $\rho g A^3$ , compared to model test results (Malenica & Molin, 1995)

# Contact us

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